

A STUDY ON MANUFACTURING OF FLANGE JOINT USED IN TRANSPORTATION VEHICLE

Harsh Kumar Sharma^{1*}

¹Research Scholar, Department of Mechanical Engineering, SIRTE, Bhopal

Abstract

Flange and flange connecting it to a commercial vehicle are processed in this investigation. Assemblies of Companion Flanges and Universal Drive shafts are also available. When the flange was brought in use in the commercial vehicles, it was seen that because of heavy friction, the part is deteriorating. To prevent this issue, a design was made which was further manufactured for accomplishing the goals of this study. When the product was ready, it was tested on CMM machine to find any type of error between the designed and manufactured product. The outcome obtained from the CMM machine showed a very small percentage of error. The new manufactured product had comparatively less amount of friction loss.

Keywords: Companion Flange; Drive shaft; Lath machine; Universal drive shaft, Industrial Manufacturing.

1. Introduction

In a vehicle with rear-wheel drive, the engine's output is sent to the differential via a drive shaft. Since “the bending natural frequency of a shaft is inversely proportional to the square of the beam length and proportionate to the square root of the specific modulus”, the driving shaft is often made in two parts to raise the fundamental bending natural frequency. It is thus preferable to utilise a one-piece stainless steel drive shaft for this application, which is why stainless steel is the material of choice. As

* ISBN No. 978-81-953278-8-1

Harsh Kumar Sharma

torque carriers, drive shafts experience torsional and shear stress equal to the difference between input torque and output torque. As a result, they must be capable of withstanding the pressure. Individual companion flanges or whole assemblies with universal drive shafts are available. For adequate torque transmission, a flanged yoke may be connected to another form of connection.

In “front-wheel drive, four-wheel drive, and the previously described front-engine rear-wheel drive systems”, drive shafts are employed differently. Other vehicles, such as motorbikes, locomotives, and maritime vessels, also use drive shafts. Drive shafts for a typical front engine, rear wheel drive vehicle are shown below (some cars have the transmission at the back).

1.1. Flange Joint

A pipeline system's components, such as pipes, valves, pumps, and other equipment, are linked together using flanges. Cleaning, inspecting, and modifying are all made much easier as a result. Most flanges are attached with screws or by welding. To create a tight seal, two flanges are bolted together and a gasket is sandwiched in between.

Flange ASME B16.5 comes in a variety of sizes. Welding Neck flange NPS 6, Class 150, Schedule 40 ASME B16.5 is referred to as such in Japan, Canada, and Australia.

It is possible to block or connect other components such as valves, nozzles, and special items using a pipe flange which is a disc-shaped piping component. After welding, piping flanges are the most popular as joining methods. Wherever, any dismantling of components is required for maintenance, inspection, replacement, or operational purposes piping flanged joints are preferred. Pipe flanges use bolts and gasket in between to ensure leakage-free piping joints. Piping flanges are selected based on pressure-temperature rating and pipe class following ASME B 16.5 or ASME B 16.47 standard. However, custom made pipe flanges can be manufactured but not preferred in industries. Piping flanges are the best alternative to welding or threading and manufactured by forging.

The most used flange types ASME B16.5 are: “Welding Neck, Slip On, Socket Weld, Lap Joint, Threaded and Blind flange”.

1.2. Process Flow Diagram

Diagrams describing the interactions between main plant components are shown in a Process Flow Diagram (PFD). Though its techniques are commonly applied to other processes as well, chemical engineering and process engineering are the most common fields in which it is used. To record a process, refine a process, or model a new one, it's utilised. If you want to name it anything else, you may call it anything from Process Flow Chart to Flow-sheet to Macro-flowchart to Top-down flowchart to Piping and Instrument Diagram to System Diagram, depending on what you want to call it and what it is used for. They represent a process using a set of symbols and notations. From crude, hand-drawn scribble to professional-looking, expanding detail diagrams, the symbols and diagrams vary in various parts of the world.

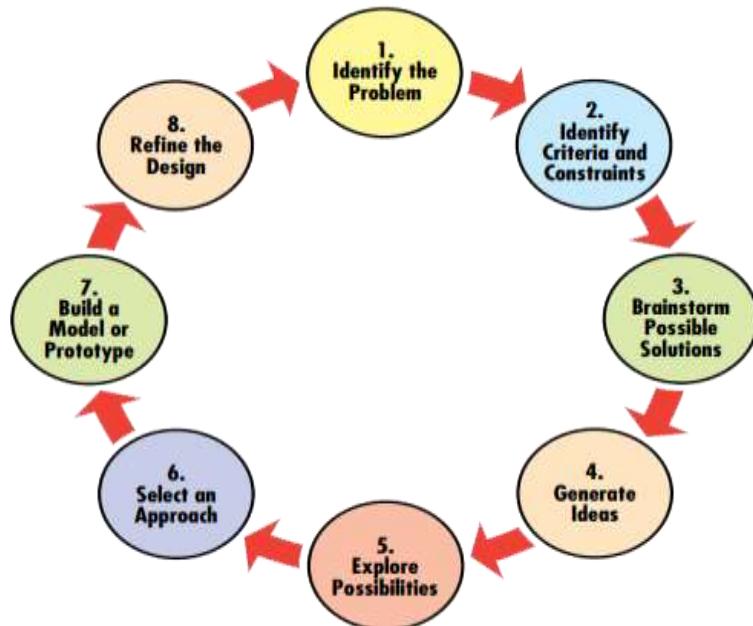


Figure 1: Process Flow Diagram

1.3. Objectives of the Study

- Analysis of problems in manufacturing process(s).
- The Design will be optimized as per analysis for DFM.
- To achieve the reduction in the non-conformance of final output.
- The jigs and fixtures will be introduced for quality improvement.
- Establishing new/advanced methods for measurement of parts.
- Creating Standard operation procedures for workstations and checking procedures.
- Better productivity will be achieved by reduction on rejection rate.

2. LITERATURE REVIEW

(Karaoglu and Sefa Kuralay, 2002) [1] By using the finite element method (FEM), a vehicle chassis with riveted joints was studied for stress. Side member thickness, connection plate thickness and connection plate length were altered to reduce stress around the riveted junction of the chassis frame.. The strains on the side member may be decreased by increasing the thickness of the side member locally, according to computer simulations. If changing the thickness isn't an option, expanding the length of the connecting plate can be. A vehicle's static and dynamic loads are the same for everybody. Inertia forces caused by driving on uneven roads contribute to dynamic loads. Because the total weight of the chassis frame rises with increasing thickness, it is necessary to keep in mind.

Harsh Kumar Sharma

(Kirkemo, 2002) [2] A wide variety of flange joints for high-pressure applications in industrial piping, pressure vessels, pipes, risers, and related equipment have been utilised widely with self seating and pressure actuated seal rings. Compact flange joints are often lighter and smaller, with smaller bolts, than regular gasketed flange joints of the same rating. To build small flange joints that can withstand pressure and external stresses, this document gives all the essential information. Also included in this document are instructions on how to design the seal ring, flange and bolts. Weld neck flanges with a homogeneous hub thickness are thoroughly examined.

(Sivakandhan and Prabhu, 2011) [3] Composite drive shafts for power transmission have been studied and optimised in this study. E-glass/epoxy and high modulus carbon/epoxy composites are used to make a one-piece composite drive shaft for automobiles. An ansys-based method for optimising the design of composite drive shafts is discussed here. As a result of the constraints imposed by torque transmission, the weight of a shaft must be minimised by employing ansys. Torsion strength, torsion buckling, and natural frequency of bending are the primary considerations. The draught shaft is designed in such a way that it is lightest and most cost-effective while yet meeting the aforementioned load criteria. Optimal laminated plate and shell designs exposed to buckling stresses and fundamental natural frequencies were evaluated. For symmetric angle-ply shells of uniform thickness, methods were suggested to determine the best ply angle variation across the thickness.

(Abel et al., 2012) [4] “Mechatronic shifting simulation of automated commercial vehicle transmissions” is utilised in Daimler's truck engineering divisions for optimization and development today. To demonstrate new functional mock-up interfaces in the ITEA2 project Modelisar, this application was used in conjunction with ITI GmbH and SIM-PACK AG (FMI). Models from a variety of different tools may be used to create the overall system for the mechanical shifting simulation by using these common interfaces. It was possible to transfer control modules from MATLAB/Simulink to a “SimulationX powertrain model using FMI for Model Exchange”, and then from the SimulationX 1D-multiphysics powertrain model to a multi-body vehicle model in SIMPACK via this method.

(Zulfadhl Bin and Zaki, 2012) [5] Transaxles are a standard feature on all cars, at least those with rear-wheel drive and a front-engine layout. If the weight reduction of the drive shaft can be accomplished without an increase in cost and a drop in quality and dependability, this is a much desired aim. Composite drive shafts may be made lighter by increasing the first natural frequency and decreasing the bending stresses of the shaft utilising varied stacking sequence. This is doable. The transmission of torque and torsional buckling capabilities are also improved by performing the same thing. “High Strength Carbon drive shafts” are being used in lieu of traditional steel drive shafts in a car. In order to reduce vehicle weight while maintaining the same level of quality and dependability, the automotive industry is turning to composite material technologies for structural component fabrication.

(Sagar R Dharmadhikari, Sachin G Mahakalkar, Jayant P Giri, 2013) [6] The focus of this research is on the evaluation of drive shaft optimization using ANSYS and the Genetic Algorithm. For the drive shaft, using a composite material rather than traditional steel gives designers more flexibility in their designs because of the material's higher specific stiffness and strength. The drive shaft is the

most important part of an automobile's drive system. Many drawbacks, such as poor specific stiffness and strength, come with using standard steel in the manufacture of drive shafts. If the design variables are not continuous, these approaches are not applicable. Structural engineering optimization, on the other hand, relies heavily on the use of discrete design factors. Constraints on building and manufacturing techniques have led to a lack of standard components.

3. RESEARCH METHODOLOGY

3.1. Steps of working

- Problem identified by using root cause analysis.
- Collect raw material for manufacturing process
- Outer Diameter & Face clean cut will be occurred in the companion flange on Lath Machine
- CNC machining
- Hardening process by using induction hardening.
- Flange spline is created by using Broaching operation
- VMC machine for maintaining the PCD
- Plunge Grinding
- Inspection report of part

3.2. Collecting raw material

First of all for manufacturing companion flange, Raw material is selected as from different type of materials. Different tests were performed on raw material for better strength and better durability of companion flange. [7] [8] Different dimensions are considered and material grade report is an important aspect that was looked according to report given by the raw material supplier. And after that it was tested in lab for confirming the grade and other parameters to choose the final material for next process. [9] [10] [11]

Table: Material collection

S.N.	CHARACTERSTIC PRODUCT	CHARACTERSTIC PROCESS	SPL CHAR CLASS	PRODUCT/PROCESS SPECIFICATION	EVALUTION/ MEASUREMENT TECHNIQUE	SAMPLE SIZE	SAMPLE FREQ.
1.	Surface			Smooth & Rustles	Visual	5 Per Lot	
2.	All Dimension			As per forging drawing	Respective gage	5 Per Lot	
3.	Material grade			EN8D/BS970	Lab report	1 Per Lot	
4.	Material grade			EN8D/BS970	Supplier report		

Harsh Kumar Sharma

4. RESULTS AND DISCUSSION

4.1. Material Testing report

Material Testing report

MATERIAL SPEC. EN8D					HEAT CODE(I/H):-AF	
Element	C%	Mg%	Si%	P%	S%	REMARKS
SPECIFD.	MIN	0.40	0.60	0.15	--	--
	MAX	0.50	0.90	0.35	0.06	0.06
ACTUAL		0.45	0.76	0.23	0.031	0.023
METALLOGRAPHIC OBSERVATIONS: -						
TEST	SPECIFICATION	OBSERVATIONS			REMARK	
Case microstructure	Fine Tempered martensite	Fine tempered martensite without ferrite			OK	
Core microstructure	Hardened and tempered	Tempered martensite			OK	
Inclusion Rating	≤ 2 ABCD IS: 4163	1.0 A,0.5B,0.5C,1.0D			OK	
Grain size	ASTM- 5 to 8	6.5– 7 ASTM			OK	
MECHANICAL PROPERTIES:-						
TEST	SPECIFICATION	OBSERVATIONS			REMARK	
Surf. hardness	500-570Hv	'560-565 Hv(53/54 HRC)			OK	
Core hardness	300-400HV	370-380 HV			OK	
Effective case depth	1.5 ± 1 mm	1.7-2.0mm			OK	
Effective case depth at oil groove	1.5 ± 1 mm	0.8mm-1.0mm			OK	
Magnafluxing/ acid etching/ visual	Shall be free from cracks, fold, seams etc.	No Cracks and free from other defects			OK	

4.2. M.P.I. Check Sheet

Technological Advancements : Research & Reviews

MPI testing report of Companion flange

4.3. CMM Report

The report was extracted from JJ precision and it showed the maximum deviation of less than 0.05mm. This deviation is under acceptable condition.

JJ PRECISION		Temperature workpiece					
T BLOCK165/1/1B MIDC BHOISARI, PUNE 411026 Email: cmi.jpprecision@gmail.com precisionwg@gmail.com							
Drawing No: * drawingno *		Date: May 16, 2021	Time: 2.29.01 pm	Order			
Operator: Master		CMM: C32Bit	Incremental Part Number: 3				
Name	ID	Aktual	Nominal	pos Tol	neg Tol	Diff	<-->
	Overall Result						
All Characteristics:	25						
- in Tolerance:	12						
- Out of tolerance:	13						
- Over Warning Limit:	0						
- Not Calculated:	0						
Total Coord. systems:	1						
- Not Calculated:	0						
Total Text elements:	0						
	Z Value_Symmetry wrt Ø12.0	-0.0076	0.0000	0.0100	-0.0100	-0.0076	-
	X Value_Slot1	-0.0145	0.0000	0.0100	-0.0100	0.0145	0.0045
	Z Value_Slot1	-0.0134	0.0000	0.0100	-0.0100	0.0134	0.0034
	X Value_Cylinder Ø14.0	-74.8844	-75.0000	0.0250	-0.0250	0.0156	-
	Y Value_Cylinder Ø14.0	-20.9712	-21.0000	-0.9600	-0.9600	0.0268	-
	Parallelism Ø14.0 wrt C	Par	0.0100	0.0000	0.0100	0.0100	-
	Parallelism Ø14.0 wrt A	Par	0.0064	0.0000	0.0100	0.0064	-
	Perpendicularity Ø14.0 wrt B	Perp	0.0277	0.0000	0.0100	0.0277	0.0177
	X Value_Cylinder Ø14.0 wrt CartDist	11.0063	11.0000	11.0030	10.9830	0.0063	0.0033
	Diameter_Cylinder3	D	14.0038	14.0120	14.0210	14.0030	-0.0082
	Diameter_Cylinder1	D	12.0026	12.0000	0.0110	0.0000	0.0026
	Width_Slot1	Width	10.0240	10.0000	10.0300	9.9940	0.0240
	Distance Ø14.0 To 12.0_ToCartDist	14.0-12.0	150.0018	150.0000	-0.0100	-0.0320	0.0019
	Distance Ø14.0 To 12.0_BeCartDist	14.0-12.0	149.9939	150.0000	-0.0100	-0.0320	-0.0001
	Y Value_Cylinder Ø12.0	Y	12.0-0.0	-20.9430	-20.0765	-20.9600	-0.0335
	Parallelism Ø12.0 wrt A	Par	12.0-0.0	0.0070	0.0000	0.0100	0.0070
	Parallelism Ø12.0 wrt C	Par	0.0003	0.0000	0.0100	0.0003	-
	Diameter_Cylinder2	D	11.9970	12.0000	12.0110	11.9980	-0.0030
	Cartesian Distance1	CartDist	-15.9945.9856	45.9310	45.9560	45.9060	0.0546
	Z Value_Symmetry2	Z	-0.0008	0.0000	0.0100	-0.0100	-0.0008
	Cartesian Distance2	CartDist	-17.9947.9866	48.0850	48.0940	48.0460	-0.0884
	Z Value_Symmetry3	Z	-0.0114	0.0000	0.0100	-0.0100	-0.0114
	Cartesian Distance3	CartDist	26.8897	27.0000	0.1000	-0.1000	-0.1003
	Cartesian Distance4	CartDist	119.2420	119.0000	0.1000	-0.1000	0.1420
	Cartesian Distance5	CartDist	29.8783	26.0000	0.1000	-0.1000	-0.1217

5. CONCLUSION

All the procedures of manufacturing have been successfully operated and the design proposed with the help of this study can be manufactured and successfully used in commercial vehicles on the results provided by this study. With the help of the CMM machine, the deviation in the proposed design and actual products was evaluated. The maximum deviation obtained was less than 0.05mm. previously, the rejection rate was 31, which with help of the present study has been reduced to 19. On the other hand, the productivity of the product has increased from 84.5% to 90.5%.

References

- [1] C. Karaoglu and N. Sefa Kuralay, "Stress analysis of a truck chassis with riveted joints," *Finite Elem. Anal. Des.*, vol. 38, no. 12, pp. 1115–1130, 2002, doi: 10.1016/S0168-874X(02)00054-9.
- [2] F. Kirkemo, "Design of compact flange joints," *Am. Soc. Mech. Eng. Press. Vessel. Pip. Div. PVP*, vol. 433, no. January 2002, pp. 91–104, 2002, doi: 10.1115/PVP2002-1087.

Technological Advancements : Research & Reviews

[3] C. Sivakandhan and P. S. Prabhu, “Optimum Design and Analysis of the Drive Shaft in Composite Material,” *Mater. Sci. Res. India*, vol. 8, no. 1, pp. 125–130, 2011, doi: 10.13005/msri/080118.

[4] A. Abel, T. Blochwitz, A. Eichberger, P. Hamann, and U. Rein, “Functional Mock-up Interface in Mechatronic Gearshift Simulation for Commercial Vehicles,” *Proc. 9th Int. Model. Conf. Sept. 3-5, 2012, Munich, Ger.*, vol. 76, pp. 775–780, 2012, doi: 10.3384/ecp12076775.

[5] M. Zulfadhli Bin and M. D. Zaki, “DESIGN AND ANALYSIS OF A COMPOSITE DRIVE SHAFT FOR AN AUTOMOBILE,” *Int. Rev. Appl. Eng. Res.*, vol. 4, no. 1, pp. 21–28, 2012.

[6] N. D. K. Sagar R Dharmadhikari, Sachin G Mahakalkar, Jayant P Giri, “Design and Analysis of Composite Drive Shaft using ANSYS and Genetic Algorithm’ A Critical Review,” *Ijmer*, vol. 3, no. 1, pp. 490–496, 2013.

[7] M. kishore, J. Keerthi, and V. kumar, “Design and Analysis of Drive Shaft of an Automobile,” *Int. J. Eng. Trends Technol.*, vol. 38, no. 6, pp. 291–296, 2016, doi: 10.14445/22315381/ijett-v38p253.

[8] P. Karthikeyan, R. Gobinath, L. Ajith Kumar, and D. Xavier Jenish, “Design and Analysis of Drive Shaft using Kevlar/Epoxy and Glass/Epoxy as a Composite Material,” *IOP Conf. Ser. Mater. Sci. Eng.*, vol. 197, no. 1, 2017, doi: 10.1088/1757-899X/197/1/012048.

[9] P. Seyfried, E. J. M. Taiss, A. C. Calijorne, F. P. Li, and Q. F. Song, “Light weighting opportunities and material choice for commercial vehicle frame structures from a design point of view,” *Adv. Manuf.*, vol. 3, no. 1, pp. 19–26, 2015, doi: 10.1007/s40436-015-0103-8.

[10] H. Mohrbacher, M. Spöttl, and J. Paegle, “Innovative manufacturing technology enabling light weighting with steel in commercial vehicles,” *Adv. Manuf.*, vol. 3, no. 1, pp. 3–18, 2015, doi: 10.1007/s40436-015-0101-x.

[11] P. N. Amrish, “Computer Aided Design and Analysis of Disc Brake Rotors,” *Adv. Automob. Eng.*, vol. 05, no. 02, 2016, doi: 10.4172/2167-7670.1000144.